

The DURLOK self-locking system is the solution to one of the most persistent problems in the field of fastener technology

...VIBRATION.

- Will not loosen or unscrew even under the most severe transverse jarring and vibration.
- Fatigue resistant design in thread runout and under-head fillet.
- Free-spinning during installation and removal.

- Re-usable with locking ability maintained.
- Embedding is no greater than with standard types of fasteners.
- Effectiveness at elevated temperatures up to 300°C is ensured.
- Closely controlled manufacturing for extra safety and reliability.

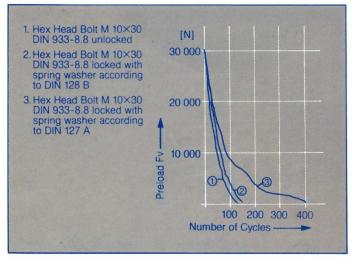
How can the self-locking ability be evaluated?

The most commonly used method for measuring locking ability has been by the indirect method of measuring and comparing the tightening and untightening torques. However, there is a growing realisation that such a test in no way simulates the self-loosening mechanics of a fastener subjected to vibration. The only way this can be achieved is to apply a vibratory force to the bolted joint and determine whether the fastener rotates loose. This has been attempted but without achieving any real measure of the self-locking ability of the fastener.

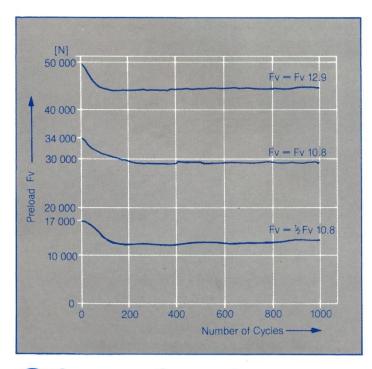
There are numerous possibilities of recording test data. However, the clearest presentation of self-locking ability is shown by recording loss of preload versus number of cycles.

A typical recording for both unlocked bolts and bolts supposedly locked with spring washers shows that the initial bolt preload is completely lost after very few test cycles; conclusive evidence that the bolt has undergone total self-loosening.

These results clearly show that spring washers do not possess any genuine self-locking ability.

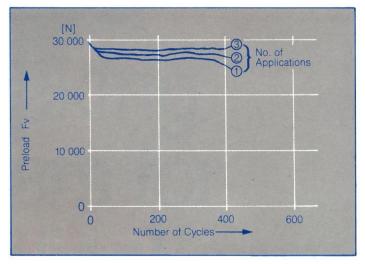


DURLOK bolts, however do not rotate loose when tested in the same way, even under the heaviest amplitudes. Even when only half of the recommended preload was used, Durlok bolts still did not loosen. This is illustrated by the figure below, which is an original recording of a vibration test on M 10 DURLOK bolts. This shows that there is minimal loss of preload even when the fastener is re-used.



Other advantages of DURLOK

DURLOK bolts and nuts are suitable for re-use because the serrations cause relatively little damage to the clamped material. This means that the locking ability can be maintained as shown by the original vibration test recording below.



This recording shows that the minimal loss of preload due to embedding even decreases due to cold-working of the surface of the clamped material during retightening of the fastener.

The DURLOK fastener system is effective on a wide variety of engineering materials including steel — both heat-treated and non heat-treated, cast irons including nodular types, non-ferrous metals and sheet materials.

Metric Sizes — Metric Units

Size	Stress area A _S	Property class	Proof load	Load at yield point α 0.2 × A _S	Load at minimum UTS	Induced preload F _{max} for μ thread	Tightening torques T _{max} (N for μ head of		
	mm²		(N)	(N)	(N)	=0.125 (N)	0.125	0.16	0.20
M6	20.1	12.9	19500	22100	24500	15950	18.2	21.0	24.0
M8	36.6	12.9	35500	40300	44600	29300	44.0	50.0	58.0
M10	58.0	12.9	56300	63800	70800	46600	84.0	96.0	109.0
M12	84.3	12.9	81800	92700	102800	68000	148.0	169.0	194.0
M14	115.0	12.9	111500	126500	140000	93000	233.0	266.0	304.0
M16	157.0	12.9	152000	172500	191500	129000	362.0	413.0	472.0
M20	245.0	12.9	238000	270000	299000	201000	698.0	797.0	913.0

Metric sizes: All dimensions in millimetres.

The presence of oil or other lubricants, organic or inorganic coatings should not adversely affect the locking ability.

DURLOK fasteners can be used at elevated temperatures up to 300°C and, in addition, can be manufactured in special materials to perform at even higher temperatures.

The value of the friction coefficient in the bearing area, μ_{H} , has a different value to that of the friction coefficient in the threads, μ_{T} , due to the serrations. As for all bolts, the friction coefficient under the head is a function of the material, surface finish and lubrication condition of the contacting materials. To account for this, the tightening torques are listed for various values of μ_{H} .

For guidance, the following chart is designed to indicate the appropriate value of friction coefficient to be applied for various engineering materials and finishes. The values of μ_{H} are based on the results of comprehensive tests.

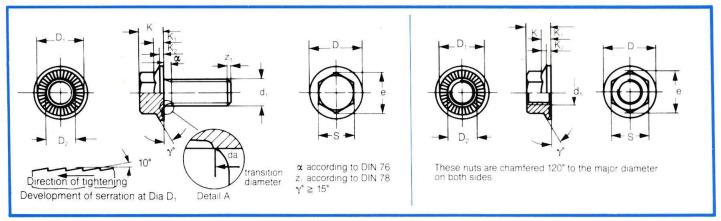
Bare Bolt	Method of Producing Bearing Surface					
Coated Surface Surface	Fine Turning Grinding	Turning, Boring, Milling	Rough Turning Rough Milling			
STEEL Hardness 250-350 HV	0.125	0.125	0.125			
STEEL Hardness 150-250 HV	0.20	0.16	0.16			
Grey Cast Iron Nodular Cast Iron	0.20	0.16	0.125			

Coatings are: Thermal oxide, phosphate, zinc plating, cadmium plating.

Notes:

For malleable cast iron bearing material, tightening torques should be obtained for grey cast iron and then increased by 10%.

Dimensions



Metric Sizes: All dimensions in millimetres

d	M6	M8	M10	M12	M14	M16	M20
D(h 15)	14	18	21	25	28	32	39
D1 min	11.8	15.2	17.2	20.6	22.8	25.7	31.2
D2*	6.6 (7.4)	9.0 (9.5)	11 (12.5)	14 (15)	16 (17)	18 (19)	22.0 (22.3)
e min*	11.10	14.49	16.64	19.01 (21.1)	21.23 (24.5)	24.62 (26.8)	30.14 (34.0)
K (js 15)	5.2	7.2	9.0	11.0	12.5	16.0 (14.5)	18.0 (18.0)
K1 max	2.27	2.70	3.24	4.00	4.50	6.38 (3.9)	6.49 (4.0)
K2	1.1	1.3	1.6	1.9	2.2	3.8 (2.4)	3.1 (2.4)
da max	6.20	8.50	10.40	13.30	15.20	17.20	21.00
S*	10	13	15	17 (19)	19 (22)	22 (24)	27 (30)

^{*} Dimensions in brackets are for nuts.

Strength grades: 12.9 for bolts and class 12 for nuts.

Material: Special alloy steel according to DIN 267 Sheet 3 for 12.9 bolts and special steels for class 12 nuts.

Thread: According to DIN 13 medium fit (6 g) for the bolts and 6 H for the nuts.

Technical Data

The induced assembly preload, F_{max} , and the corresponding tightening torques, T_{max} , are based on a 90% utilisation of the minimum yield strength by combined tension and torsional stresses. For cases where the yield strength must never be exceeded during tightening, the tightening torque must be reduced by a value equivalent to the scatter. Comprehensive investigation has shown that the scatter, due to variations in friction coefficient and torque scatter when tightening by torque wrench, must be accounted for by using a reduced torque T which is 90% of the tabulated value T_{max} , $T = 0.9 \times T_{max}$.

Accordingly, the induced preload $F_{max}\,will$ be reduced to the new preload F $_{F}\,=\,0.9\,\times\,F_{max.}$

It should be noted that preload and tightening torque are a function of the joint stiffness. The tabulated values are valid for a joint stiffness which occurs under snug conditions with a clamping length of 2.5—4d. In addition, the values are based on an average friction coefficient for the threads of $\mu=0.125$.

This data also applies to DURLOK nuts when used with 12.9 strength grade DURLOK bolts.

Where other design criteria exist, consult the supplier for technical advice.

Mechanical properties of DURLOK fasteners

DURLOK bolts:

Mechanical Property	DURLOK 12.9
Tensile Strength, Rm (N/mm²) min	1220
0.2% Proof Stress (N/mm²) min	1100
Stress under Proof Load Sp (N/mm²)	970
% Elongation 5D min	9
Core Hardness HV. max	420

DURLOK nuts:

DURLOK nuts are manufactured to meet the mechanical properties of ISO class 12 and are suitable for use with DURLOK bolts